

DNS of Turbulent Combustion in Complex Flows

UK Consortium on Turbulent Reacting Flows Annual Meeting Jacqueline H. Chen jhchen@sandia.gov



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DNS of Flame Propagation and Structure Behind a Backwards Facing Step

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Motivation



Early injection PCCI



Gain insights into:

- Flame stabilization mechanism
- Effect of heat release
- Turbulence-chemistry interactions





S3D - multiblock



Backward-facing step



- Mechanism: 22 species non-stiff reduced ethylene-air (Lu et al. 2012)
- Transport model: mixture averaged
- Turbulent inflow profile: feed data generated from a separate 3D DNS of channel



Results

Vorticity magnitude



Heat release rate



Vorticity dynamics

Enstropy



• Enstrophy present dominantly on product side

Vorticity dynamics



Flame stabilization



C = 0.05

= 0.8



Flame structure



- Flame structure varies with distance from the step
- Flame-flame interaction present due to the shear
- Incomplete oxidation

Flame structure







Flame structure



Summary

- DNS of C2H4/air flame stabilization behind a backwards facing step
- Strong interaction between recirculation zone, shear layer, and flame brush
- Radicals from the recirculation zone assist in anchoring the flame
- Turbulence generation migrates towards products downstream of the stabilization point
- Implications for modifications to flame structure and heat losses to the wall

Direct Numerical Simulation of flame stabilization assisted by auto-ignition at *reheat* conditions

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- Originally developed by ABB for high efficiency, load flexibility and low emissions
- Recently improved and simplified (reduced cost) for the H-class GT36
- First (premix) combustion stage based on flame propagation
- Second (sequential) combustion stage based on auto-ignition



 \bullet Adjusting firing temperature of 1st stage allows control of $t_{\mbox{\scriptsize ign}}$ in 2nd stage

Hydrogen fuel

- Flashback in 1st stage
- Early auto-ignition in 2nd stage



- 2nd stage is mainly auto-ignition stabilized
- 2nd stage inlet temperature needs to be decreased and not 2nd stage flame temperature

Hydrogen fuel

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- 2nd stage inlet temperature needs to be decreased and not 2nd stage flame temperature
- 1st stage de-rating is compensated by shifting fuel to 2nd stage

Reheat burner

DNS of idealized reheat burner configuration from Ansaldo Energia Operating conditions:

- Inlet temperature: ~ 1100 K
- Pressure: ~ 20 atm

Scaled conditions:

- Mean inlet temperature:
- Pressure: 1 atm
- Fuel: hydrogen

Ignition length (L_{ign}) Turbulent NSCBC inflow Isothermal walls

Objective:

- Understand the flame stabilization
- Identify the modes of combustion
- Quantify the role of auto ignition

S3D - Multiblock

- Mildly complex geometry enabled by multi-block DNS capability
- Construct geometries by assembling several cuboidal blocks (like Lego)
- Compressible formulation (J. H. Chen et al., CSD 2009)
- Spatial derivatives: 8th order CD schemes & 10th order filter
- Time integration: 4th order Runge-Kutta
- Mixture averaged transport model



Weak scaling on Titan

Simulation details



1.25 billion grid points 20 million CPU hours $Re_b = 13000$

- Chemical mechanism: 9 species hydrogen-air (Li et al., 2004)
- Inflow composition: premixed H2 + O2 + N2 + H2O (ϕ = 0.35)
- Ubulk = 200m/s, u' = 20m/s, T_{inlet} = 1100K, T_{wall} = 750K
- Inflow profile: feed from DNS of a fully developed channel flow





Two combustion configurations are observed:

- Design state: mainly auto-ignition in the combustion chamber
- Intermittent auto-ignition state: ignition in mixing section

Design combustion state

Heat release rate



Mass fraction of HO₂



Combustion modes:

- Autoignition along centerline
- Flame propagation near corners
- HO2: indicative of chain branching

Transport budget analysis

$$\frac{\partial (\rho Y_{OH})}{\partial t} = -\nabla_{\beta} \cdot (\rho Y_{OH} \mathbf{u}_{\beta}) - \nabla_{\beta} \cdot (\rho Y_{OH} \mathbf{V}_{\beta,OH}) + W_{OH} \dot{\omega}_{OH}$$
Advection Diffusion Reaction

- Auto-ignition: balance between advection and reaction
- Flame propagation: balance between diffusion and reaction





Chemical Explosive Mode Analysis

• $\alpha = \phi s / \phi \omega$: ratio of the projected non-chemical source term and the projected chemical source term (C. Xu et al., PROCI 2018)

Three mode are identified:

- Assisted-ignition ($\alpha > 1$): diffusion significantly promotes reaction
- Auto-ignition ($-1 < \alpha < 1$): chemistry plays a dominant role
- Extinction zone ($\alpha < -1$): diffusion dominates chemistry and suppresses ignition



Intermittent auto-ignition state

Heat release rate



Mass fraction of HO₂



- Early auto-ignition in the mixing section
- Ignition kernel advects downstream
- Occurs intermittently

Intermittent auto-ignition state



Contours of heat release

- Local rise in pressure
- Increases local temperature by 20-30 K
- High reactivity of hydrogen
- Decrease in ignition delay time (30%)

Conclusions

- Performed DNS of a reheat burner at scaled conditions
- Two states of hydrogen/air combustion have been observed:
 - design state: flame propagation and auto-ignition in the combustor
 - intermittent auto-ignition in mixing section
- Premature auto-ignition arises due to pressure (and following temperature) rise in mixing section
- Quantified the contribution of different modes towards heat release using chemically explosive mode analysis (CEMA)
- Future work:
 - characterize the unstable flame behavior and the conditions leading to it
 - find the inlet conditions for statistically stationary reheat flame
 - perform 2D and 3D simulations with varying fuel composition and its stratification

ECP has formulated a holistic approach that uses codesign and integration to achieve capable exascale



ECP's work encompasses applications, system software, hardware technologies and architectures, and workforce development

From Paul Messina's ASCAC talk April 19, 2017



30 Exascale Computing Project, www.exascaleproject.org

ECP application: Transforming Combustion Science and Technology Through Exascale Simulation (Pele)

Pele: Block-structured adaptive mesh

CFD Simula of RCCI

DONE



S3D: multi-block compressible reacting DNS multi-physics validation: spray, soot, radiation

Argonne BERKELEY LAB VALUE AND VALUE

The Pele Project

Transforming Combustion Science and Technology with Exascale Simulations



- Use exascale platforms to solve first of exascale-era combustion problems
- Anchored in basic research needs: requirements driven by gas phase chemical kinetics research questions
 - turbulence chemistry interaction in conditions motivated by IC engine research
 - provide a path for development of scalable design codes suitable for exascale hardware



Challenge Problem - Motivation

Advanced combustion regimes (LTC)



PCCI/HCCI – load limitations Requires precise charge preparation and combustion control mechanisms (for auto-ignition and combustion timing) Conventional diesel *



Early injection PCCI



Adapted from content provided by University of Wisconsin Engine Research Center

UNIVERSITY OF WISCONSIN - ENGINE RESEARCH CENTER

High Fidelity DNS and Hybrid DNS/LES of RCCI Diesel Combustion

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		Characteristic / Need	Approach		
		Impulsively started jets with disparate scales between fronts and turbulence. (Outer scales: 10cm, ms Inner timescales: microns, ns)	Dynamic adaptive mesh refinement		
		High speed injection followed by subsonic conditions downstream	Compressible and low-Mach capabilities		
	-Soot-precursor -Nonpremixed -Lean HTC -Rich HTC -LTC	Long time horizons to set up turbulence for studying fundamental TCI	Hybrid DNS/LES [Non-reacting LES, DNS for flame]		
		Lean, rich, and low temperature chemistry critical in multi-stage ignition and formation of soot precursors.	Accurate and detailed thermochemistry.		
		Liquid fuel injection	Lagrangian spray model		
		Coupling between mixture preparation and emissions	Detailed kinetics including emissions, sectional model for soot with radiation		
DNS of a ndodecane		Mixture preparation dependent on re-entrainment of combustion products	Realistic piston dish and cylinder wall geometry		

DNS of a ndodecane spatially evolving turbulent diesel jet flame at 60 bar, combustion modes, Dalakoti et al. 2018

Performant on exascale architecture

Design Philosophy, Strategy

- The Pele suite:
 - Compressible (PeleC) and low Mach number (PeleLM) integrators, compatible design, data, I/O, etc
 - Shared physics (ideal/real gas kinetics, thermodynamics, transport, sprays, etc) PelePhysics
 - Block-structured adaptive mesh refinement, built on **AMReX** framework
 - Robust, accurate, extensible finite-volume (conservative) discretizations
 - Embedded boundary treatment of arbitrary geometries
 - Enables CAD-to-compute, avoids expensive, time-consuming difficult mesh generation step
 - AMReX-supported X+Y parallelism (inter-node X and intra-node Y)
 - Parallel mesh and particle data, specialized to needs of AMR (temporal subcycling, etc), plus Fork-Join type temporary redistribution strategies, high-performance I/O and insitu/in-transit analysis support
 - Combustion-specific agile code generation: Fuego+SINGE for GPU-optimized CUDA kinetics evaluation, Kokkos-based kernels for particle/fluid coupling terms
- Agile development framework, open source modular design, continuous integration/testing
- Close interaction with AMReX for new capabilities in development
 - App-relevant GPU/A21 implementation of AMReX structures/algorithms
 - Leverage AMReX multi-app development

Pele Code Design Overview

- Baseline algorithm design for multicomponent flow with stiff reactions, AMR
 - PeleC: Comparable advection, diffusion time scales, motivates IMEX-type scheme based on Spectral Deferred Corrections (SDC) with *time-implicit* chemistry
 - Robust highly efficient time-explicit Godunov-type upwind advection, simple centered diffusion
 - BDF-style implicit chemistry ODE integration, with sources that approximate the other processes
 - **PeleLM**: acoustics filtered away analytically, but still want robust, time-explicit advection
 - Chemistry *and* diffusion are now *time-implicit iterative timestep* simultaneously incorporates flow constraint (constant pressure), mutually coupled species/energy diffusion and chemistry. Entire system evolved stably on slower advection time scales across AMR grid hierarchy
- SDC-based iterative timestep treats each process essentially independently, with accelerated iteration to couple everything nicely together
- Robust baseline allows stable, well-behaved *extensible* time step
 - Switch 2nd order advection scheme with more accurate 4th order algorithm
 - Option for "destiffened" chemistry model that allows highly efficient time-explicit advance
 - Robust to other, potentially stiff, tightly coupled processes, such as sprays, radiation, soot, etc

Project Roadmap

	2017	2018	2019	2020	2021	2022
Model Development	Hybrid LES/DNS	Deep learning SGS model training methodology	Rare event detection		In situ training	
Compressible Capabilities	Non-ideal, single level sprays static EB geometry	Non-reacting LES, 🗸 AMR sprays 🗸	Hybrid LES/DNS	Soot	Radiation	Moving EB geometry
Capability Towards Challenge Problem	60 Bar open, gas phase dodecane low mach jet	Atmospheric pressure, compressible static geometry flows	60 Bar open, ideal gas dodecane low mach jet wall impingement	60 Bar open, dodecane jet with hybrid multi-physics	60 Bar compressible + low Mach, re-entrainment	
Low Mach Capabilities			Static EB geometry	Sprays, Hybrid LES/DNS	Hybrid compressible / LM Non-ideal EOS	
Automated Thermochem	Initial demonstration	Pressure independent kinetics	Mechanism generation	Reduced dual-fuel mechanism		
Performance and software	PeleC Exists Understand potential of task parallel	PeleLM tiling	GPU performance Multi-physics load balancing	Performance portability		A21 / Frontier Performance Enhancements



DNS of Ndodecane Multi-injection Jet at diesel conditions -Parameters

- N-dodecane/air injection with $Y_{NC12H26} = 0.45, 446K$
- Jet: D=0.17mm, U=30m/s, Re=15,000
- Environment: 60 bar, 900K, 15/85% O2/N2 ('spray A')
- 10 micron resolution
- 53 species mechanism
 - Age variable tracks the fluid age/residence time

Diffusion coefficient based on Le=1 for all add. scalars

$$\frac{\partial \rho a}{\partial t} + \frac{\partial \rho u_j a}{\partial x_j} = \frac{\partial}{\partial x_j} \left(\rho D \frac{\partial a}{\partial x_j} \right) + \rho$$













Comparison at 0.4ms after pilot/main injection start

Pilot: 0.26ms, dwell: 0.17ms, main: 0.74ms left: times w.r.t pilot start (0.4ms total time), right: times w.r.t. main start (0.83ms total time)





Comparison at 0.4ms after pilot/main injection start

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Comparison at 0.4ms after pilot/main injection start



Difference in pilot and main ignition - temperature



Pilot:

- -Stoich. conditions close
- to homogeneous ign.
- -Difference for rich cond.

Main:

-Significant differences on rich side

Pilot: 0.26ms, dwell: 0.17ms, main: 0.74ms; left: times w.r.t pilot start, right: times w.r.t. main start





Difference in pilot and main ignition – OC12H23OOH

Pilot: 0.26ms, dwell: 0.17ms, main: 0.74ms; left: times w.r.t pilot start, right: times w.r.t. main start

