

Low temperature combustion in ignition of turbulent dual fuel mixture

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1.1 Motivation

- Lean premixed combustion of natural gas (NG) in IC engine is attracting to reduce particulate and NOx emission, with poor ignitability.
- Dual-fuelling, ignition of natural gas (NG) by a high-cetane fuel (e.g. DME), presents promise to ensure successful combustion initiation control.
- Dual-fuel combustion involves combined processes of autoignition, diffusion flames, and flame propagation, which is challenging for experimental measurement and numerical modelling.



Few fundamental numerical studies of dual-fuel ignition:

• Ignition in laminar heptane/CH4-air mixing layer (Wang et al. 2015).

- Heptane droplets ignition in CH4-air (Demosthenous et al. 2016).
- Pilot ignited DME/CH4-air (Soriano & Richardson)
- Dynamics of triple flames in igniting DME/CH4-air (Tai Jin, Kai Luo et al. PCI, 2018)
- Understanding of the ignition dynamics in dual-fuel mixture is of great importance for design and control.

1.2 Objectives

LTI to cool flame?

Accelerate or not?

Autoignition in diesel engines occurs as a multi-stage process, involving Low- and Hightemperature combustion (LTC & HTC)

- Single-stage HTI in turbulent mixture no earlier than HID ($\tau_{HT,mr}$) (Mastorakos, 2009) LT ignition (LTI) transits to propagating cool flame:
- Accelerate HTI, earlier and richer than τ_{HT} (Borghesi et al. (2015), Krisman et al. (2016, 2017), Borghesi et al. (2018))
- No accelerated HTI, vary between 2 and 3 times τ_{HT} in n-heptane jet (Krisman et al. (2017))



2.1 Ignition in dual-fuel mixture



Configuration 1: **Pilot-ignited** DME/CH4-air L=3.2mm P=40atm CH4/Air, Ø=0.3 Isotropic turbulence T=1050K oeriodic DME. T=450K periodic ξ∈(0,1) Single ∇ ξ Tai Jin et al., PCI (2018)





2.2 Numerical Schemes

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Governing equations: Continuity + Navier-Stokes + Species/Energy transport + EOS

A fully compressible in-house DNS code

1. Spatial discretization: Eighth-order central differencing scheme

$$f_{i}' = \frac{8/5(f_{i+1} - f_{i-1})}{2h} - \frac{4/5(f_{i+2} - f_{i-2})}{4h} + \frac{8/35(f_{i+3} - f_{i-3})}{6h} - \frac{1/35(f_{i+4} - f_{i-4})}{8h}$$

2. Temporal discretization: Fourth-order Runge-Kutta method (RK44)

3. Filtering: Explicit tenth-order filter method (suppress high wavenumber errors) $\widehat{f}_{n} = \frac{1}{2^{10}} \Big[772 f_{n} + 210 \big(f_{n-1} + f_{n+1} \big) - 120 \big(f_{n-2} + f_{n+2} \big) + 45 \big(f_{n-3} + f_{n+3} \big) - 10 \big(f_{n-4} + f_{n+4} \big) + \big(f_{n-5} + f_{n+5} \big) \Big]$

4. Boundary conditions

- Improved Navier-Stokes characteristic boundary conditions (NSCBC)
- 5. Parallel strategy: MPI communication + CPU decomposition

2.3 Chemical mechanism

 Reduced by Dr. Tianfeng Lu from a validated detailed DME/CH₄ mechanism (Mech_56.54) by Prof. Curran (2015)

- 25 species in 147 reaction steps
- Validated in ignition delay time, perfectly stirred reactor (PSR) extinction residence time, laminar premixed flame speed



2.4 Homogeneous Ignition Delay



Stoichiometric mixture (Pure CH4, Pure DME, CH4+DME)+air

With DME added,

Ignition delay (ID) decreases significantly,

NTC effects.

CH4-air (φ=0.6) → +DME

- One stage ignition \rightarrow two stages
- T_o increase, ξ_{MR} for the two stage ignition increase
- τ_{st} decrease significantly



3. Pilot-ignited dual fuel combustion

periodic





3.1 Ignition case 1



HRR (W/m³), at 0.008, 0.06, 0.16, 0.23, 0.25, 0.29, 0.29, 0.33 and 0.41 ms $\,$

Case 1, ϕ =0.3, ξ_{max} =1.0, L_t=0.2 mm

- First stage ignition (LT) transits to propagating cool flame
- High T ignition kernel discretely located in fuel rich mixture

- Triple flames propagating along the ξ_{st} line
- Lean premixed branch initiates the premixed methane-air flame

Tai Jin, Kai H. Luo et al. Dynamics of triple-flames in ignition of turbulent dual fuel mixture: a direct numerical simulation study, PCI, 2018.

3.2 Ignition case 2



Case 2, φ=0.6, ξ_{max}=1.0, L_t=0.1 mm

The ignition and flame initiation process is qualitatively consistent with $\varphi=0.3$

Temporal evolution of HRR (W/m³), t*=0.8, 1.2, 1.6, 2.0.

3.3 Ignition case 3



Time evolution of HRR (W/m³), t*=0.864, 1.0, 1.2, 1.2.

Case 4, φ=0.6, T_o=1300K

 The ignition process is qualitatively consistent with Cases 1&2

 High T ignition kernels much bigger



3.4 HT ignition kernels

 10^{-2} T_o=1300K Sst HIKs ξ_{st} **time (s)** 10⁻³

- 0.6 x10⁻³ 10⁻² T_o=1050 K u'=0.5 1.5 t. u'=1.0 cool flame 0.5 u'=0.5 u'=1.0 **time (s)** st 0.4 0.3 10⁻⁴ 0.5 0.2 م 0.2 0.3 0.1 0.2 0.4 0 0.3 0.4 ع No obviously accelerated accelerated
- 10-4 0.2 <u>8</u> 0.3 0.1 0.5 0 0.4

- HIKs located in a wide • range of ξ (fuel rich)
- HIKs are accelerated •

- Most of HIKs located around ξ_{mr}
- HIKs in fuel rich mixtures are accelerated
- Velocity fluctuation, no significant change time of HIKs

4. Ignition in turbulent dual-fuel mixture

case	Т_о (К)	ξ	ξ′	ξ _{MR}	ξ _{st}	ID @ ξ _{MR}	1 st ξ _{MR}	1 st ID
1	1050	0.4	0.025	0.19	0.0413	2.3e-4	0.14	8.71e-5
2	1050	0.2	0.015	0.19	0.0413	2.3e-4	0.14	8.71e-5
3	1300	0.4	0.025	0.046	0.0413	1.62e-4	0.289	7.29e-5

Computational parameters





4.1 Ignition-case 1



Case 1, $\overline{\xi}$ =0.4

First stage ignition (LT) transits to propagating cool flame

 High T ignition kernel discretely located in fuel rich mixture

4.1 Ignition-case 1



Case 1, $\overline{\xi}$ =0.4

• High T flames connected

- No obvious propagating triple flames
- Lean premixed branch initiates the premixed methane-air flame

4.1 Ignition-case 1

The role of diffusion in supporting the cool flame, transport budget analysis for the LTC marker $Y_{CH3OCH2O2}$ Ignition front: R>>D, Propagating flame front: R \simeq D (R-reactive, D-diffusion terms)



A.B:
 R>>D, ignition front,

 χ relatively low

C-F:
 R&D increase, R≃D,
 cool flame front

4.2 Ignition-case 2



Case 2, ξ=0.2

 Cool flame exists around low T mixture (high ξ, larger 1st stage ignition delay)

- HRR, Y-CH₃OCH₂O₂, relatively high around cool flame
- High T ignition occur in a large area, not in discrete kernels

4.2 Ignition-case 2



Case 2, $\overline{\xi}$ =0.2

• High T flames emerge with each other

• High T flames connected, propagate across ξ_{st} , and initiate the premixed methane-air flame

4.3 Ignition-case 3





Case 3, $\overline{\xi}$ =0.4, T_o=1300K

E11

0

E11

20

10

50

- Cool flame develops both in the central region and in the mixing layer
- HTI first initiate along ξ_{st} , around single-stage ξ_{mr}
- third-stage ignition kernels

4.4 HT ignition kernels





Time of HIKs compared with homogeneous ignition delay times

Case 1, $\overline{\xi}$ =0.4

HIKs in fuel rich mixture, accelerated Comparable with $\tau_{\text{HT},mr}$

Case 2, $\overline{\xi}$ =0.2

HIKs in fuel rich mixture HT ID shorter than the shortest $\tau_{\text{HT},\text{mr}}$

ID of CH4/air: 6.98ms Premixed CH4/air flame initiation time: Case 1: 0.34ms Case 2: 0.26ms

4.4 HT ignition kernels

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ξ∈(0,1) **Single** ∇ ξ



ξ∈(0,0.4) **Various** ∇ ξ



5. Conclusions



Ignition dynamics in turbulent DME/methane-air mixture via DNS

- The ignition process involves both LTC and HTC, varies with the thermochemical conditions, as well as turbulence
- Low temperature combustion plays a vital role

LTI to cool flame?

• Depends on the mixture fraction gradient (ignition delay gradient)

Accelerate or not?

- High-T ignition in fuel quite rich mixture is accelerated by passage of cool flame
- High-T ignition can be shorter than the shortest τ_{HT,mr}



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Questions or comments?

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