## Study of thermoacoustic instabilities in premixed hydrogen-enriched swirling flames using LES

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### **LES of Hydrogen Enriched Flames**

#### Backgrounds

- Lean premixed combustion often exhibits thermoacoustic and hydrodynamic instability
- $H_2$  combustion exhibits higher laminar flame speed, higher flame temperature and lower lean flammability.

#### Target Flames

- Hydrogen enriched flames based on PRECCINSTA burner studied experimentally in DLR (Chterev, 2019).
- Technically premixed, swirl stabilized.
- Selected Operating conditions

Equivalence ratio	0.85
Thermal Power	23 kw
%Vol.H2	0%, 20%, 40%



PRECCINSTA premixed swirl burner (Chterev, 2019).

### **Numerical Setups**

#### Numerical setups

- In House compressible LES code: BOFFIN-LESc (Fredrich, 2020)
- Non-reflective outflow boundary conditions at chamber outlet (Yoo, 2005)
- Isothermal wall boundary condition
- Multiblock structured mesh ~2.7 million cells, including air plenum, swirler and combustion chamber
- Mesh independent study done in the same geometry in previous work (Fredrich, 2020)

# Air Plenum Chamber

A view of the computational mesh.

#### **Chemical Kinetics**

• 15-step reduced mechanism based on GRI-Mech 3.0 (Lu, 2008)

#### Numerical details

- Initialised for 6 flow-through times is enough for a statistically stationary flow to develop
- Statistic time-averaged over 6 flow-through times

### **Iso-thermal flow field**



Mean velocity fields and profiles at 4 downstream positions: (a) axial mean velocity (b) radial mean velocity.



RMS velocity fields and profiles at 4 downstream positions: (a) axial RMS velocity (b) radial RMS velocity.



### Flame topology

Time averaged Heat release rate (HRR), line of sight integration



Mean Heat release rate in 3 cases investigated

Hydrogen addition results in

- Higher heat release rate on average
- Shorter flame, closer to the combustor inlet
- More flash back, less lift off

p'(kPa)

P1

### **Acoustic fluctuations**

#### Self-sustained limit cycle oscillations



Heat release rate (left) and axial velocity (right) in phase

### **PSD of P', FFT of HRR compared to EXP**



- EXP p' with estimation of damping wall effect ~13dB (Lourier,2017)
- First peak frequencies matches Helmholtz f, thermoacoustic oscillations
- Over predicted amplitude: both in C1 (8dB) and C3 (20dB)
- Coupled peak f of p' and HRR in C1 and C3
- Harmonics of dominant f in C1 and C3, subharmonic and its multiples in C1

### **Velocity field**



- Strong fluctuations coupled with pressure oscillations
- Observed in all three cases
- Mean velocity field deviation:
  overpredicted jet velocity,

back flow in the centre region,

narrower jet

### Next Step

#### Why so strong instabilities?

- Relative to the coupling of flame and pressure fluctuations: not observed in isothermal case
- Varies with operation conditions c

#### Next Step Work

- Extend the computational domain
- Different operating conditions study



### Study of Noises Generated by Compositional Perturbations using LES

### Introduction

#### Two categories of combustion noises in combustors (pressure fluctuations)

- Direct noise : caused by volume expansion due to unsteady heat released
- Indirect noise: unsteady heat released also generates temperature, compositional and vortical perturbations, which if accelerated can eventually generate acoustic noise

#### Indirect noises caused by compositional disturbances

- Incomplete premixed flames, fluctuations in equivalence ratios
- Target case: Compositional and entropy indirect noise generated in Cambridge Entropy

Generator (CGW) Rig with non-isentropic convergent nozzle (Domenico, 2021)



A schematic layout of the experimental configuration.

Dimensions for the experimental configuration.

### **Direct and indirect noise generation**

#### One dimensional noise generation



- Pulse injection: last for 10ms, repetition rate: 0.25Hz (every 4 seconds)
- Acoustic wave travel speed: c; entropic and compositional wave travel speed: u
- Convective time  $\tau_c = L_c/u$ : time interval between generation of direct and indirect noise
- Reverberation: acoustic signals reflected in a acoustic chamber repeatedly in a short period of time (Rolland, 2018)

### **Direct and indirect noise generation**

#### Specific Objectives

- Direct and Indirect noise generation: probe in the upstream pipe
- Compare effect of different injection gases
- Validate the capability of BOFFIN-LESc in predicting combustion noises

#### Test Cases: 2 gases, 2 injection positions (convective lengths)

Case	Gas	$\dot{m}(gs^{-1})$	$\dot{m}_g(gs^{-1})$	$L_c(m)$
C1	He	8.0	0.17	0.65
C2	He	8.0	0.17	0.05
C3	$CO_2$	8.0	1.62	0.65
C4	$CO_2$	8.0	1.62	0.05



#### Numerical Setups

- Computational Domain covers upstream pipe, convergent nozzle and 2.1m of downstream pipe
- Fully reflective inlet boundary, non-reflective outlet boundary
- Mesh contains about 0.4m celss, clustered in nozzle and injection region
- Pressure signal phase averaged over 2 cycles

### **Pressure fluctuations at upstream probe**



Long configuration (C1 and C3)

- 0-t<sub>p</sub>: P<sub>d</sub> generation
- $t_p-t_c$ :  $P_d$  decay (decay fit line by reverberation model)
- $t_c-t_c+t_p$ : P<sub>i</sub> generation



- C1 and C3 have Similar direct noise amplitude
- Indirect noise: C1 negative, C3 positive caused by different gas molar mass compared to air
- Under predicted indirect noise amplitude in C3, good predicted  $t_{\rm c}$

### **Pressure fluctuations at upstream probe**



- Compositional entropic wave convection and dispersion
- Flow across nozzle not resolved: Mach number under predicted

### Next Step

- More data for phase averaged results
- Resolve nozzle flow: indirect noise generation, noise reflection at the nozzle
- Study compositional and entropic wave propagation in the upstream pipe, effect on indirect noise generation
- Downstream probe pressure signal



Time series of Helium mass fraction in C1.



### **References**

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