

Spectral behaviour of the heat release rate in swirl-stabilized and bluff body flames

Ankit D. Kumar¹↑, James C. Massey¹, Zhi X. Chen^{1*}, Nedunchezhian Swaminathan¹

¹ Department of Engineering
Hopkinson Laboratory

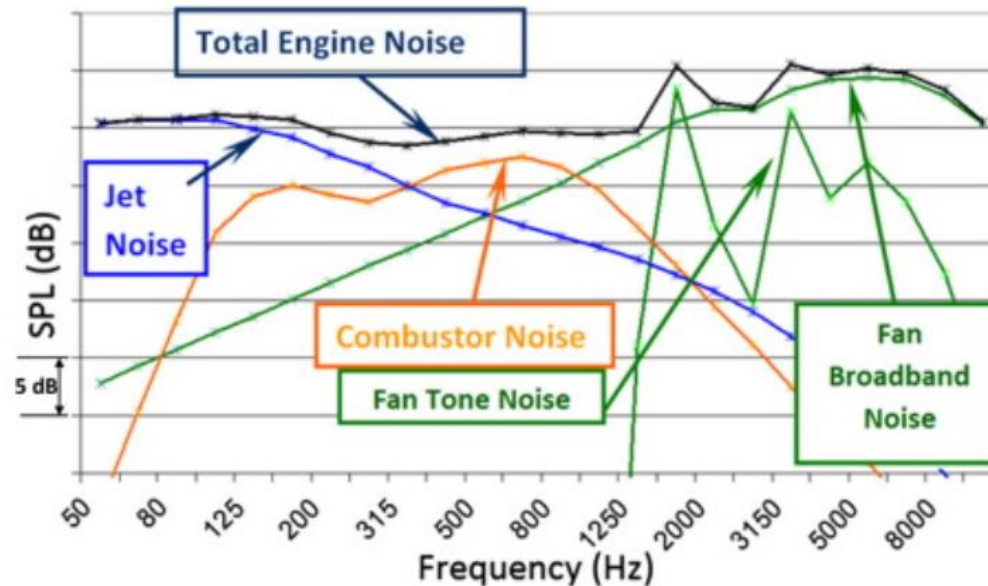
*Peking University

↑Corresponding Email-address : adk46@cam.ac.uk

Outline

- Motivation
- Background and Aim
- Description of cases
- Results
- Modelling and discussion
- Conclusion

Motivation

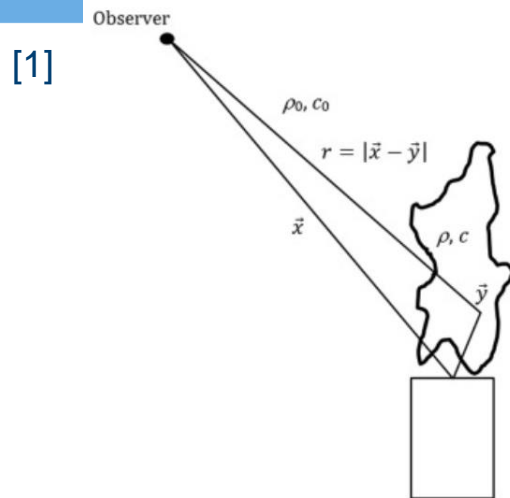


[1]

- Combustion noise is dominant for combustors operating under lean premixed/stratified conditions.
- Combustion noise has a broadband nature and is comparable to other sources.
- Noise is an important design criteria that can be predicted using LES.

1. Dowling, A. P., & Mahmoudi, Y. (2015). Combustion noise. *Proceedings of the Combustion Institute*, 35(1), 65-100.

Background and Aim



Acoustic Wave Equation [2,3]:

$$\frac{1}{c_o^2} \frac{\partial^2 p'}{\partial t} - \frac{\partial^2 p'}{\partial x_i^2} = \cancel{\mathcal{T}_1} + \cancel{\mathcal{T}_2} + \cancel{\mathcal{T}_3} + \mathcal{T}_4 \rightarrow \frac{(\gamma - 1)}{c_o^2} \frac{\partial \dot{Q}'(\mathbf{y}, t)}{\partial t}$$

$$\widehat{\mathcal{P}}(\mathbf{x}, \omega)$$

Low order models

High Fidelity LES

$$\widehat{\mathcal{P}}(\mathbf{x}, \omega) \approx \frac{\gamma-1}{c_o^2} \omega^2 \int_{V_f} |\widehat{G}_1(\mathbf{y}, \mathbf{x}, \omega)|^2 \psi_q(\mathbf{y}, \omega) V_{\text{cor}} d^3 \mathbf{y}$$

$$\widehat{\mathcal{P}}(\mathbf{x}, \omega) = \frac{(\gamma - 1)}{16 \pi r^2 c_o^2} Y_{f,u}^2 H^2 \mathcal{B}_{int}^2 \omega^2 \widehat{\Omega}_{int}(\omega) \left(\int_{V_f} \overline{\omega(\dot{\mathbf{y}}, t)} d^3 \mathbf{y} \right)^2$$

PSD of canonical open flames depends on volume integrated heat release rate (studied previously[4,5])

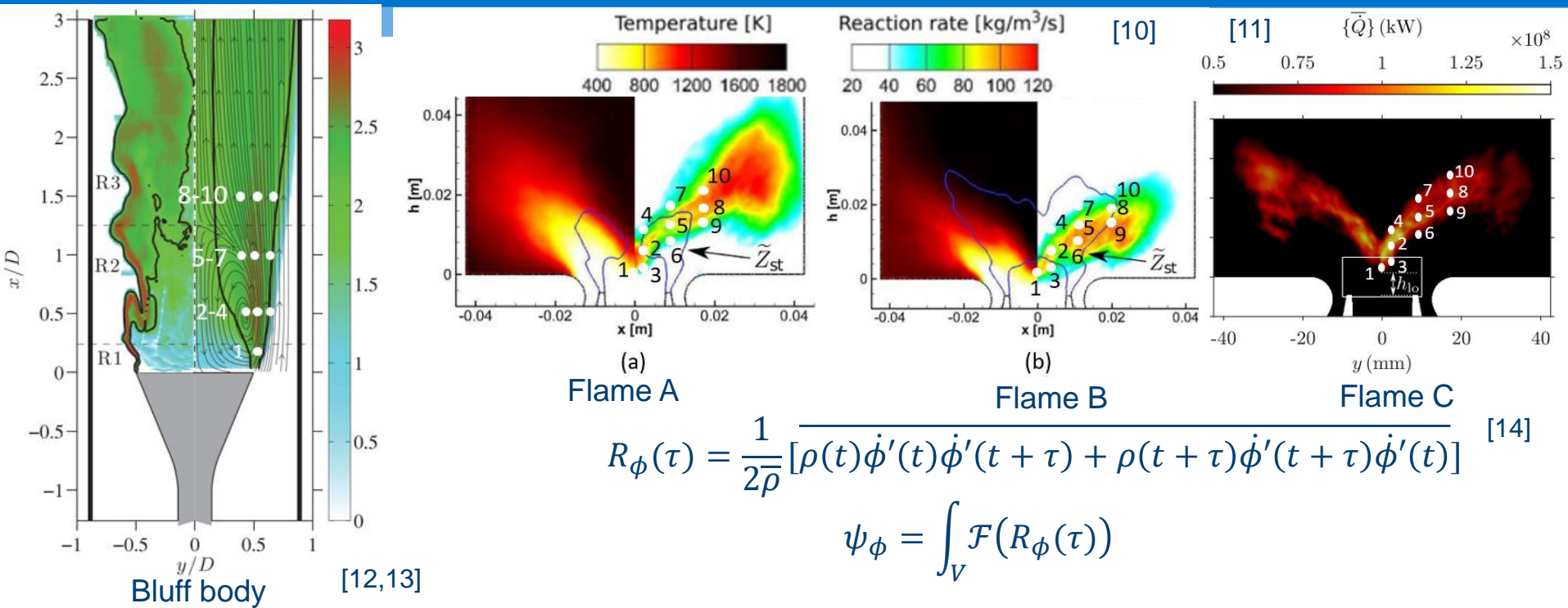
$\widehat{\mathcal{P}}$ depends on local heat release rate for closed flames seen in practical applications – Focus of this work

- ψ_q is simple to obtain in LES compared to experiments.
- ψ_q from non – reacting flow LES ?



Case	ϕ_{glob}	$U_b(\text{ms}^{-1})$	u'/s_L^o
Bluff body	0.59	15	30.46

Computational resources and data processing



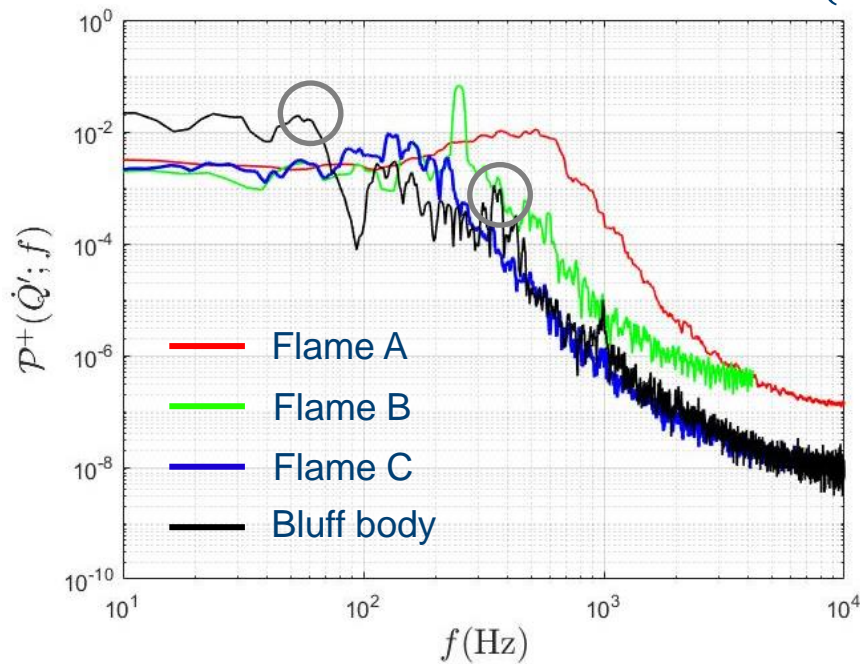
Case	$\delta t(s)$	$\mathcal{T}(s)$	N_s	f_{N_s}
Flame A	5×10^{-5}	0.055	20000	10000
Flame B	1.2×10^{-5}	0.099	8333	4166
Flame C	7.5×10^{-5}	0.3	13333	6666
Bluff body	5×10^{-5}	0.2	20000	10000

Code(s) used: OpenFOAM v2.3, v7 / FlaRe

Time usage on ARCHER (approx. CUs): 19,375.00
(since 2019 which resulted in 7 publications)

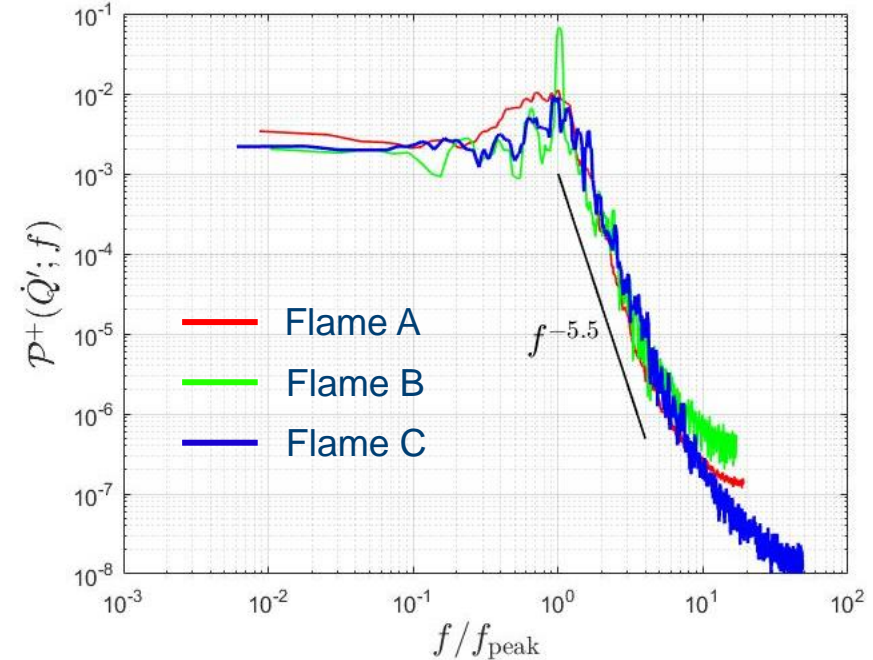
Spectrum of $\mathcal{P}^+(\dot{Q}'; f)$

$$\mathcal{P}^+(\dot{Q}'; f) = \frac{\psi(\dot{Q}'; f)}{\int \psi(\dot{Q}'; f) df}$$



$$f_{\text{peak}} \approx \frac{U_b}{L_{\text{flame}}} \quad [14]$$

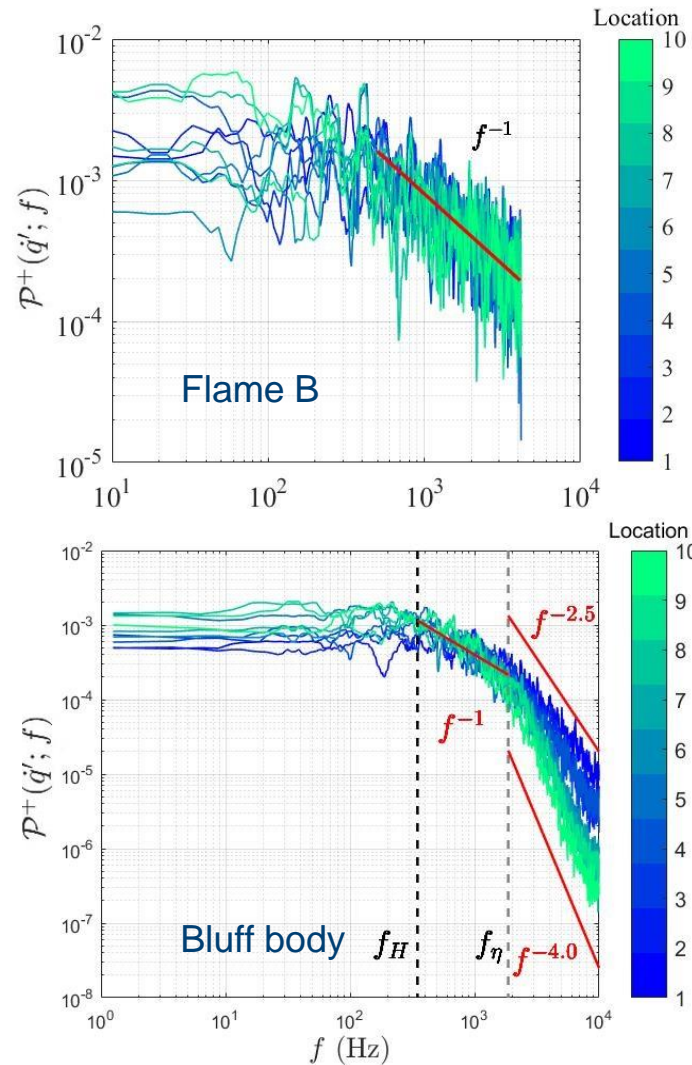
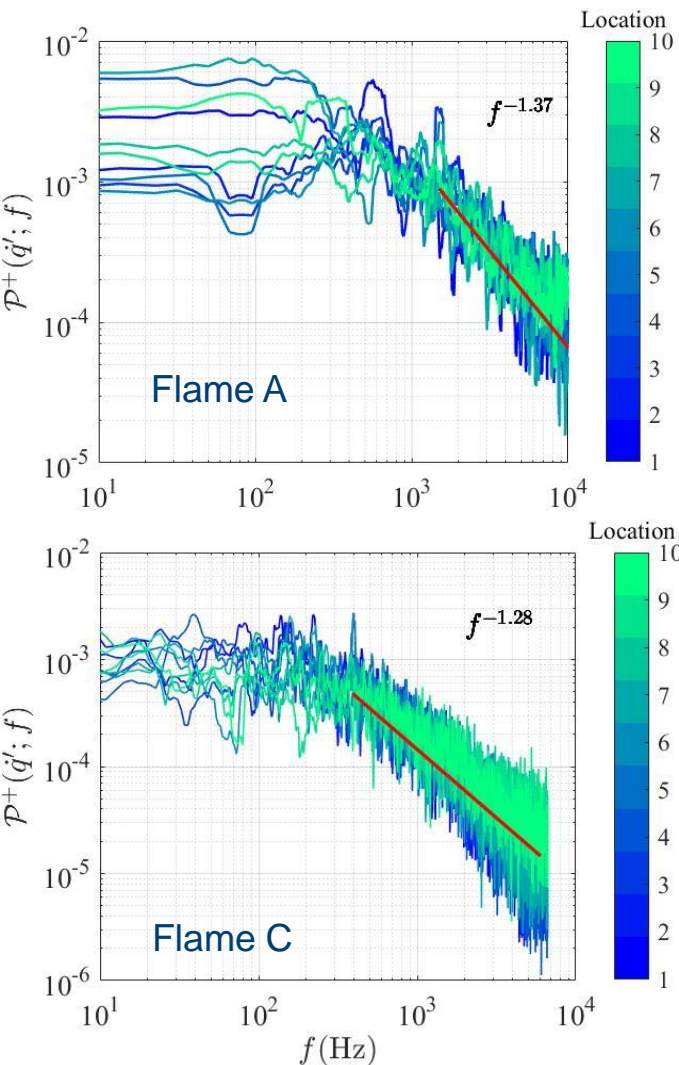
At high frequencies, $\mathcal{P}^+ \sim f^{-\frac{5}{2}}$ [15]



$$f_{\text{peak}} \approx \frac{\dot{V}}{V_{\text{flame}}} \equiv f_{\text{conv}}$$

At high frequencies, $\mathcal{P}^+ \sim f^{-5.5}$

Spatial variation of $\mathcal{P}^+(\dot{q}'; f)$

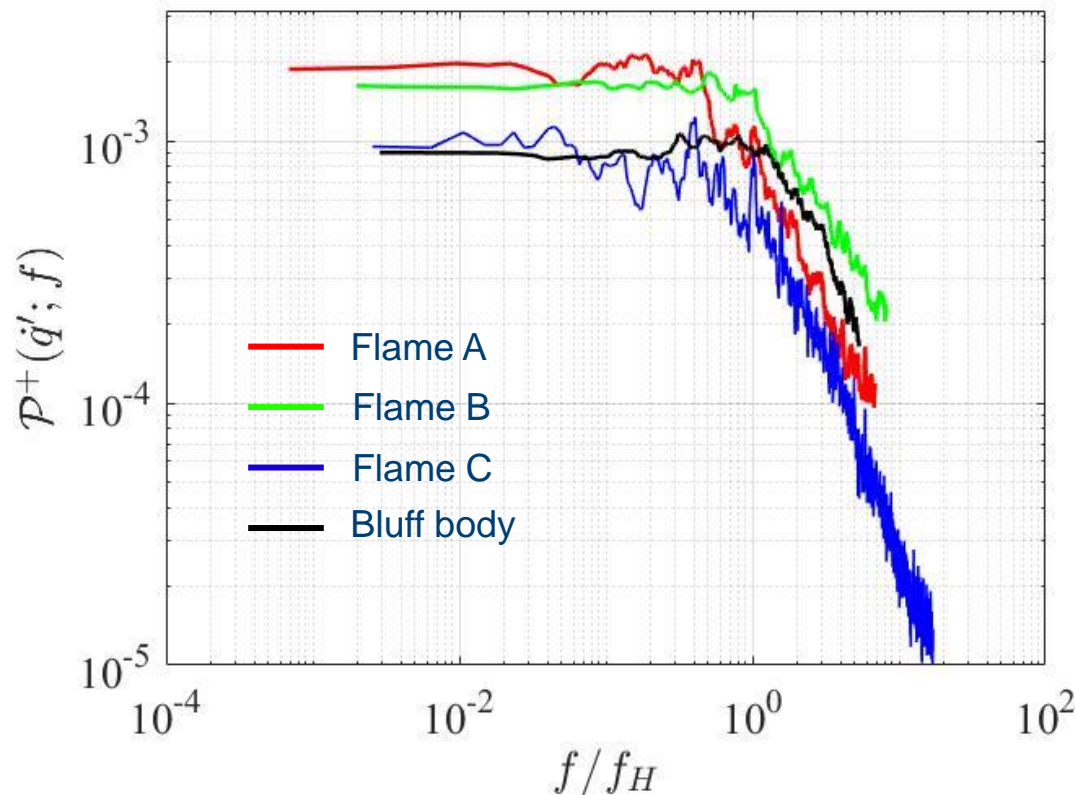


$$\mathcal{P}^+(\dot{q}'; f) = \frac{\psi(\dot{q}'; f)}{\int \psi(\dot{q}'; f) df}$$

[16,17]

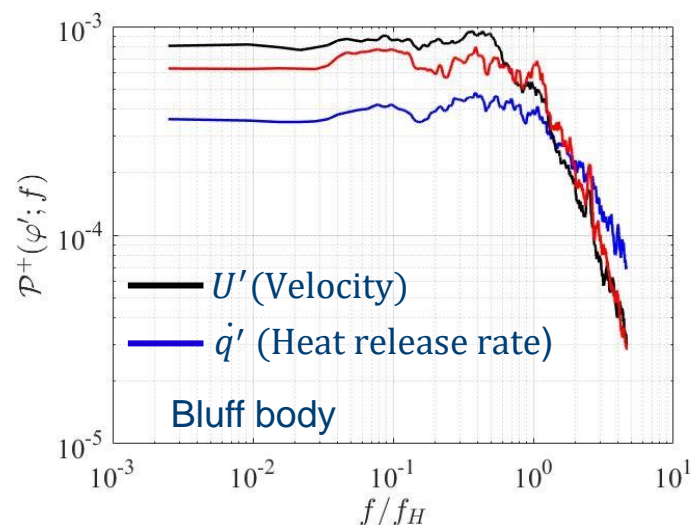
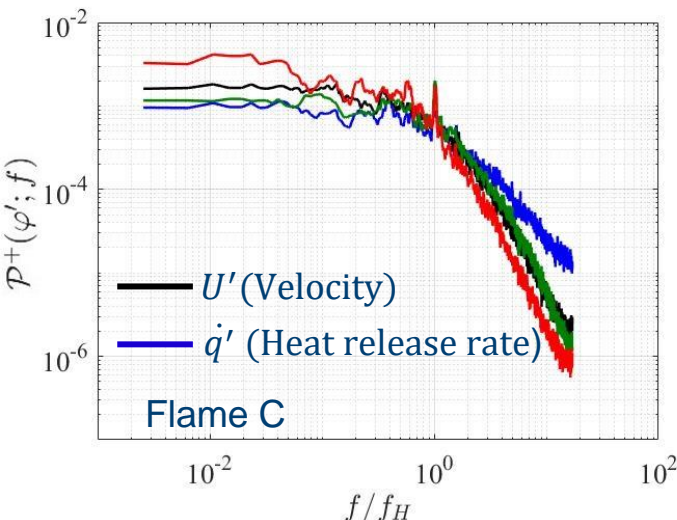
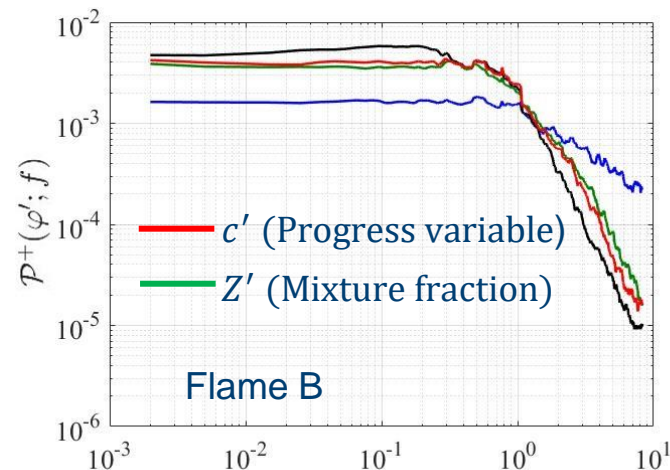
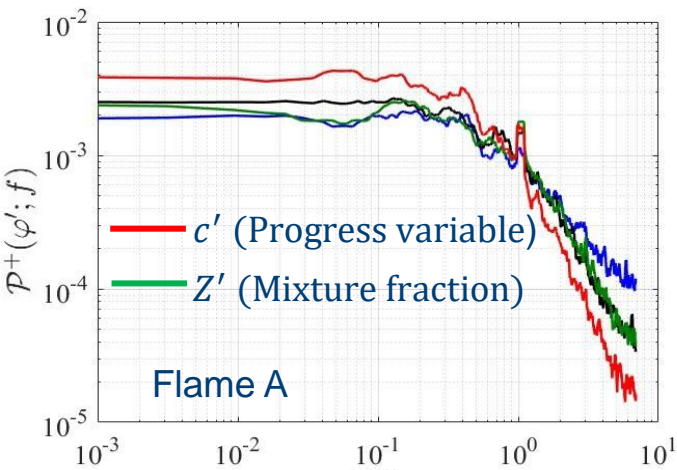
Spatial variation of $\mathcal{P}^+(\dot{q}'; f)$ – Mean spatial variation

$$\mathcal{P}^+(\dot{q}'; f) = \frac{\psi(\dot{q}'; f)}{\int \psi(\dot{q}'; f) df}$$



- Near constant behaviour at low frequencies.
- Roll off at higher frequencies at the rate : $f^{-\alpha}$; $\alpha = 1 \sim 1.37$.
- Reasonable frequency scaling with PVC and vortex shedding frequency.
- For CH_4 -air mixture the order magnitude of $\mathcal{P}^+(\dot{q}'; f)$ is similar irrespective of geometry.

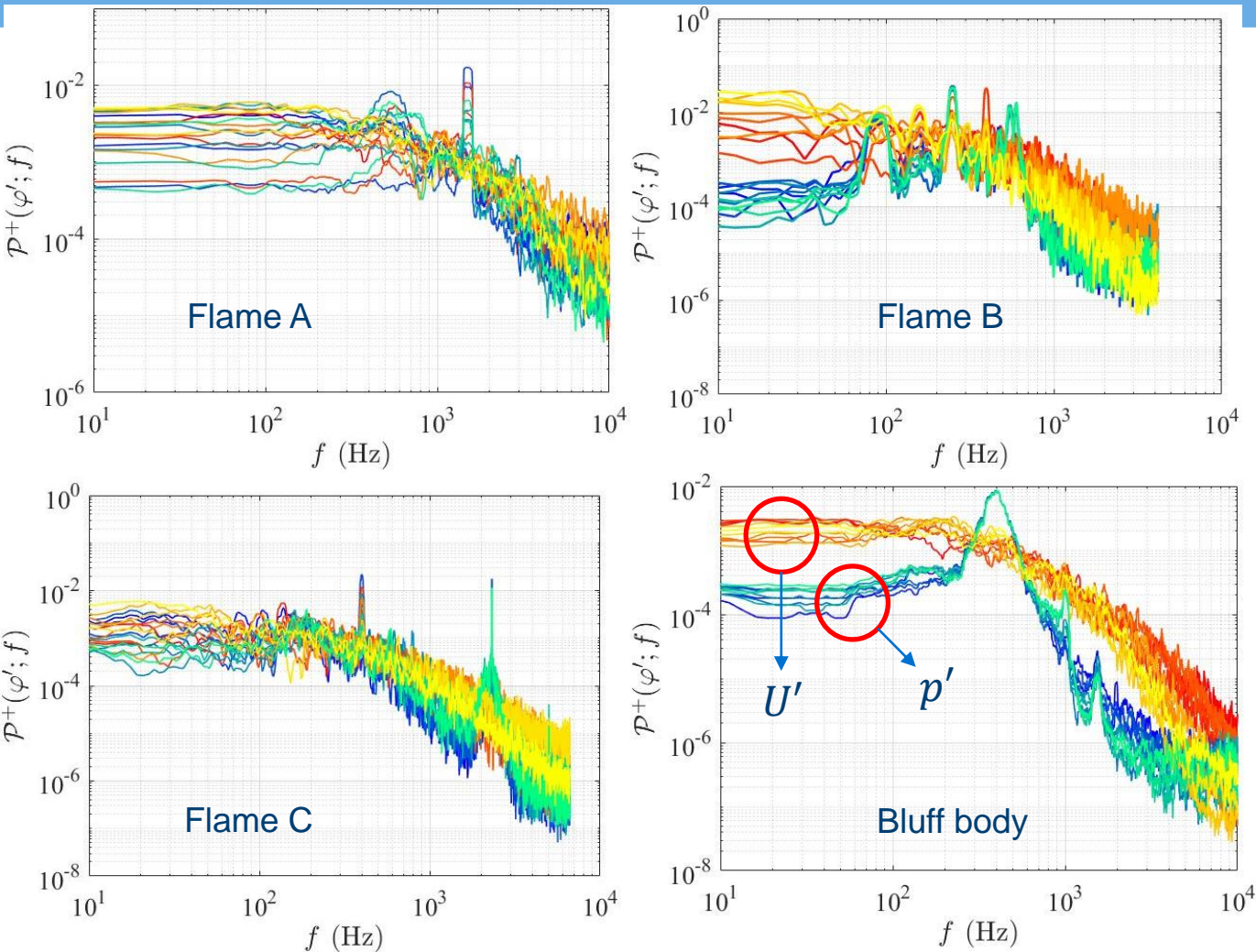
Relation to turbulence and scalar spectra



$$\mathcal{P}^+(\dot{q}'; f) = \frac{\psi(\dot{q}'; f)}{\int \psi(\dot{q}'; f) df}$$

- Flame A and C show close similarity to turbulence spectra.
- Like flame A and flame C, flame B and bluff body flames show a change in behaviour at f_H , but turbulence spectra starts to roll off at $f < f_H$.

Pressure-velocity coupling

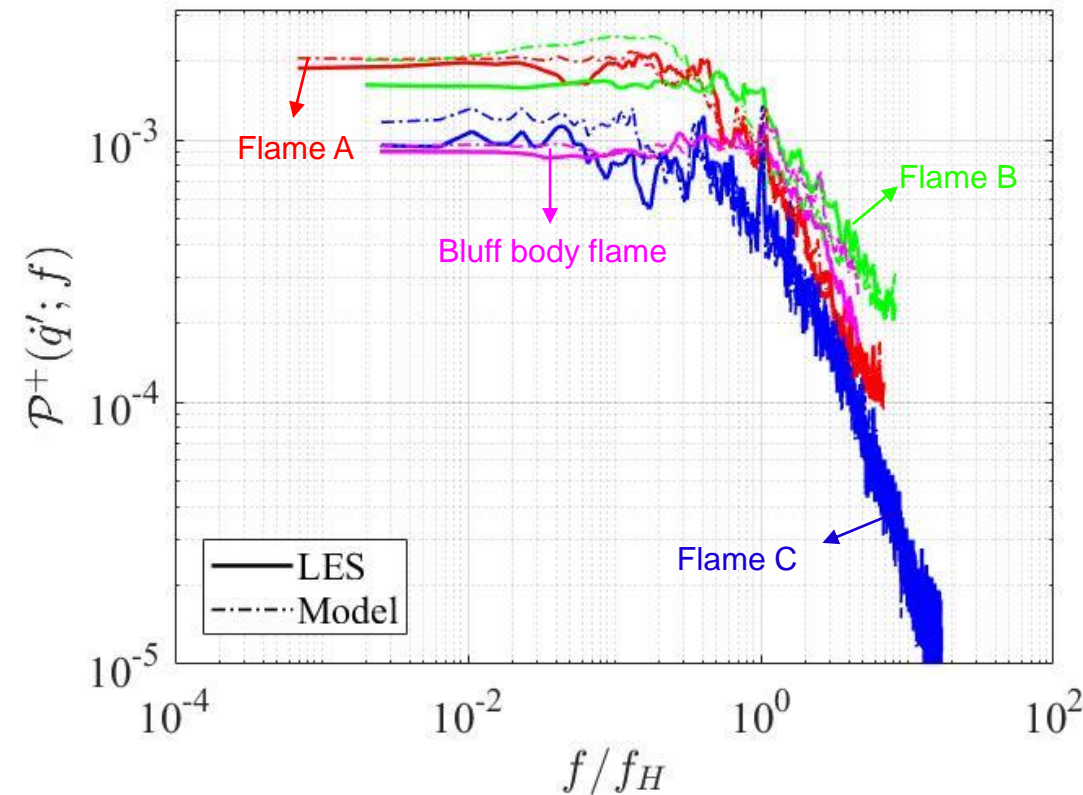


$$\mathcal{P}^+(\dot{q}'; f) = \frac{\psi(\dot{q}'; f)}{\int \psi(\dot{q}'; f) df}$$

- p' and U' in Flame A and C are strongly coupled and show similarities in spectral behaviour.
- Flame B and bluff body flames show significant deviations in low frequency regime.

Comparison of LES and model based on turbulence spectra

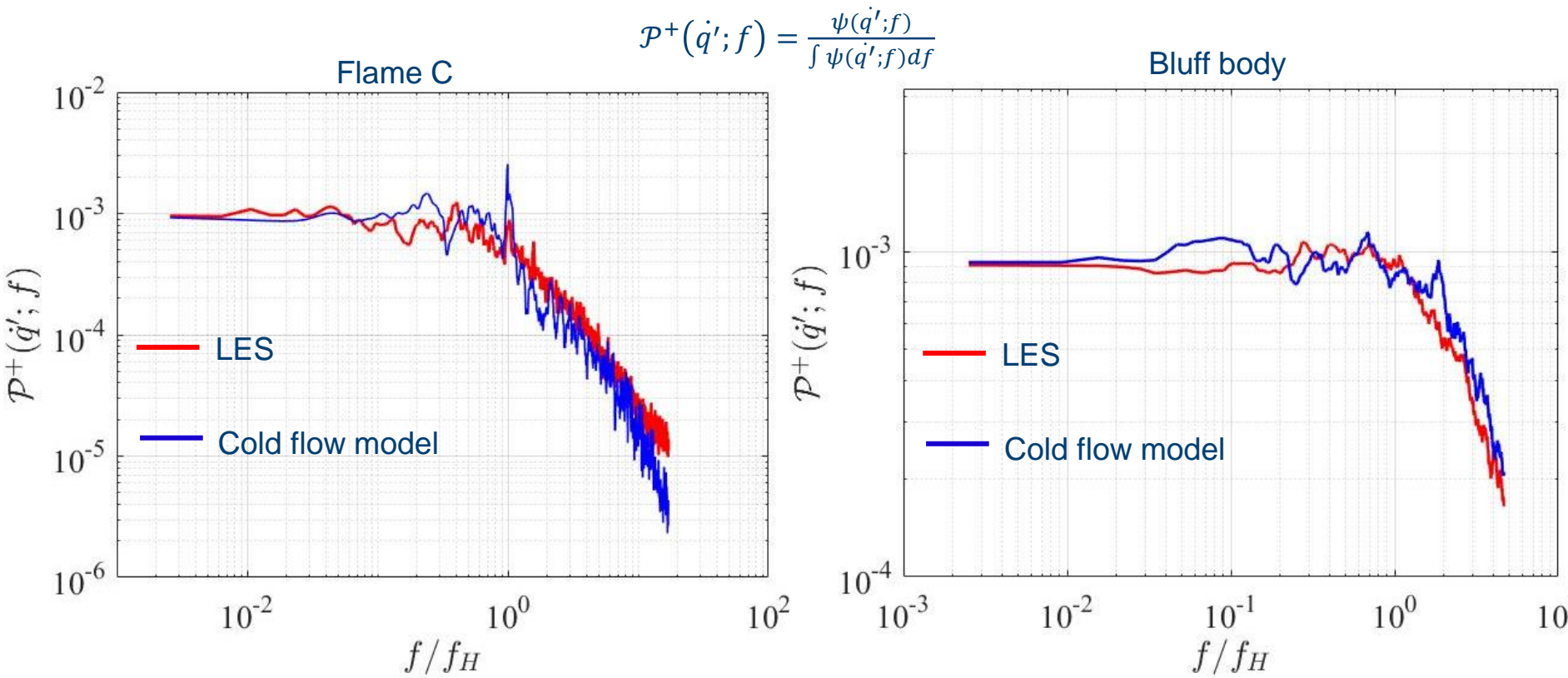
$$\mathcal{P}^+(\dot{q}'; f) = \frac{\psi(\dot{q}'; f)}{\int \psi(\dot{q}'; f) df}$$



$$\frac{\mathcal{P}^+(\dot{q}'; f)}{\mathcal{P}^+(U'; f)} = \begin{cases} k_{1U}, & f/f_H < 1 \\ k_{1U} \left(\frac{f}{f_H} \right)^{k_{2U}}, & f/f_H \geq 1 \end{cases}$$

- $\mathcal{P}^+(\dot{q}'; f)$ of Flame A and C show better agreement compared to flame B and bluff flames because of stronger correlation with hydrodynamic fluctuations.

Comparison of $\mathcal{P}^+(\dot{q}'; f)$ – LES and cold flow model ($\mathcal{P}^+(U'; f)$)



Conclusions

- Spectra of volume integrated heat release rate(HRR) shows a -5.5 fall off rate at high frequencies as opposed to -2.5 predicted by previous theoretical work, and a convective frequency scaling is still prevalent with a modified definition.
- Spectra of local heat release rate is qualitatively similar in all flames with minimal spatial variation and shows a reasonable scaling with hydrodynamic frequencies.
- A similar qualitative trend is observed for the spectra of heat release rate, velocity and other scalars under reacting conditions.
- A model relating the velocity spectrum to HRR spectrum under reacting and non-reacting conditions is proposed.
- This model shows a good agreement for various (swirling and non-swirling) flames studied using LES.

Thank you for listening. Any questions?

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